Slab Tipper

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Portable Slab Tipper

By

Thomas Durand
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Abstract:

Stone countertops are very heavy and fragile. When installing a finished countertop, up to six workers are required to lift the stone onto the countertop, risking their safety and possibly breaking the stone. The safety of both the workers and the stone are very important, so there needs to be a better way. NSI Solutions came up with an idea for a mechanism to aid in the insulation process while limiting the number of workers required, and keeping the workers safer while adequately supporting the stone to prevent it from breaking. Designing methods and calculations were done to ensure the mechanism can support the stone while limiting the size and weight of the mechanism. During the construction process several changes were made to the design of the mechanism to make it easier to construct, limit cost, and increase the strength of the Slab Tipper. During the testing process, the mechanism will be evaluated and changes will be made to improve the Slab Tipper and ensure that it will work in the field. In the end the Mechanism will consist of two separate stands that can be stored in the installation truck when not in use. Then the stands can be set up next to the cabinets that the countertop will be installed on.

Key Words: support, strength, safety
Introduction

Description:

When installing a finished slab of granite, it is crucial that the slab is supported properly. Granite is so heavy that it can break under its own weight. This could lead to loss of thousands of dollars in materials, it could damage the cupboards, or injure workers. By designing a mechanism for installing the finished counter tops, the risk of injury or loss of revenue can be greatly reduced. The mechanism will also limit the amount of workers required for a large install, reducing the labor costs for some installations.

Motivation:

A company by the name NSI Solutions has requested an insulation mechanism after their partner company broke a large slab of granite during an installation over the summer. Luckily no one was injured and they were able to fix the counter top after several additional hours of work on the counter top. This may not be true for every instant in the future.

Function Statement:

The mechanism must be able to adequately support a finished slab of stone and transport it to the countertop.

Requirements:

- Must be able to be carried by one person
  - Weigh less than 50lbs
- Must be able to support a finished countertop with a maximum weight of 1500lb
  - Stone must not flex more than .25” (number provided by NSI Solutions)
- Must be able to be operated by 2-3 person(s)
- Must protect the floors and cabinets
  - Properly disperse the weight of 1500lbs through its feet to prevent from breaking tile or wood floors (exert less than 250lb at any given point)
- Must cost less than $1000
- Must be able to be used on all counter heights
  - Height adjustable form 30” – 42”
- Must be able to be used in a 42” aisle (36” is ideal)
- Stand must be able to be setup close to cabinet
  - Pivot point 10”-12” from cabinet
Success Criteria

Success of the Slab Tipper will be judged based on its ability of demonstrating a working prototype for testing and evaluation. The Slab Tipper will be considered a success if it can successfully transport a completed stone countertop to the top of a cabinet without breaking, damaging the floor/cabinet, and reducing the amount of workers required for installation by at least one.

Design & Analysis

Installation of stone countertops requires several workers to transport the stone to the top of the counter and the stone can break under its own weight. The slap tipper is a mechanism that limits the amount of workers required to do an installation, while adequately supporting the stone reducing the risk of the stone breaking. In appendix A.1 there is a sketch showing the distribution of the mass of the stone in the upright position, for a free standing mechanism with four legs and feet, where the stone will be prepped for installation. In the upright positng the stone will be exerting a 600lb force downwards on the stand, where the pivot point will have to counteract that with a 600lb force upward to have \( F_y = 0 \). Looking at the requirement that the stand must distribute the weight of the stone to prevent a tile floor from breaking. Appendix A.2 shows the first RADD calculations with free body diagrams breaking down the forces in the stand. With a SF of 2.0 each foot of the stand exerts a force of 375lb onto the floor. The tiles are rated for 250lb of force at one point according to ASTM C648. To avoid breaking or damaging the tile a foot was designed to distribute the force onto the tile into a larger area. With the foot designs the ceramic tile should only experience 29.84 psi at any given point of the foot. The drawing for the foot can be found in Appendix B.1.

After consulting with the owners of NSI Solutions, the four leg design was scraped and a three legged design was introduced. Still taking into account that the stand has to adequately distribute the weight of the stone to prevent from damaging the floor a second RADD was calculated. In Appendix A.3 shows the new 3 legged design and the force distribution that goes along with the new design. The final force was found to be a 500lb vertical point force being exerted on the floor. Taking the 250lb point breaking force, provided by ASTM C648, it is found that any foot with a surface area will distribute the force enough to prevent the floor from breaking. Taking the findings in Appendix A.3, a ladder style foot was designed to allow for the foot to pivot on an axis to help prevent high spots. It was also decided that the surface that comes into contact with the floor will be covered in a soft rubber to reduce the chance of scratching the floor as well as adding another method to prevent high pressure points. The Ladder style foot design can be found in Appendix B.6, with a base of 5”x2”. Appendix A.17 shows that the ladder foot has a surface area of 10in^2 and with a force of 500lb on every foot, each foot will exert a pressure on the floor of 50psi.
The Slab Tipper must be able to support a slab of granite up to 1500lb without breaking the stone. Looking at the requirements the stone must not deflect more than 0.25 inches. Taking into this requirement several methods were designed, to support the center of the stone. The original design can be seen in Appendix B.3 and Appendix B.15. This method of supporting the middle of the stone used 1” diameter steel pipe that is easily purchased in a local hardware store and one or two notched beams that lock onto the pipe and can be slid into place to support the center of the stone. The issue with this design is that the maximum deflection was found to be 5.5 inches, which is significantly over the requirement, calculations for this method is in Appendix A.10.

After consulting with the owner of NSI Solutions, the design to support the middle of the stone was simplified into a rectangular tube on edge, this will provide an increased moment of inertia limiting the deflection in the middle of the stone. The new design can be seen in Appendix B.12 and the calculations can be found in Appendices A.11-13. In A.11 calculations for a 2”x1” tube made of aluminum (aluminum was chooses to try and limit the weight of the full mechanism) with a thickness of .125 inches has a deflection of 0.868inches when loaded with a 1500lb slab, this is still over the requirement. It was decided that since the middle supports won’t always be used, it isn’t important that they be made of aluminum to reduce weight. Revision 3 uses the same dimensions as the aluminum square tube but in steel. The calculations for this is in A.12, when loaded with a 1500lb slab the middle of the stone will deflect 0.289 inches, this is just slightly over the requirement. In Revision 4, the final revision, a 2”x1” steel tube is still being used but the thickness was changed to 3/16 of an inch. When loaded up with a 1500lb slab Revision 4 of the middle support deflects 0.2166 inches in the center. This is under the requirement for deflection and is decided that the middle support will be 2”x1”-3/16” Steel Tube. The calculations for Revision 4 can be found in Appendix A.13.

During the construction process there was a few concerns raised. One being the amount the Pivot Plate would deflect under a max load of 1500 lbs. Shown in the calculations in appendix A.21, the deflection of the Plate was calculated to be 0.154 in using 1.5in x 1.5in x 3/16in square tubing. This was expectable being less than the requirement of 0.25in or less. The second concern raised was the shear force in the bolts holding the mounded bushings to the pivot plate. The original .25in bolts were found to be perfectly fine for this application, the shear force in the 1/4in bolt was calculated to be 6754.54lb/in^2. This is well below the allowable shear force of a grade 8 1/4in bolt of 91ksi, the calculations for the shear force can be seen in appendix A.20, but it was decided to go with Grade 8 5/16in diameter bolts as a safety precaution as well as visual appeal, to appear stronger.

**Methods & Construction**

The main goal of this project is to make a working prototype of a slab tipper mechanism that could be developed and changed at a later date to fit extra needs and requirements to be placed on the market to sell. The main focus is the tipping mechanism needing to adequately support the stone without breaking and being able to distribute the weight evenly to prevent from damaging the floor. Building the mechanism will require a combination of specially machined parts specifically for this mechanism as well as some premade parts purchased from sellers like
McMaster-Carr and Everett Steel. Though the preferred material for the selling model would be aluminum, to limit cost of the mechanism, ease construction methods, and access of material, the prototype will be mainly constructed of steel. The reason the prototype will be made of steel is because the prototype will only be used for proof of concept of the mechanism, also steel is easily accessible and reasonably cheap for this application. It also comes in a wide variety of thicknesses and shapes to meet the needs of the mechanism. Having steel being the main material for the mechanism, it allows parts to be welded without special equipment that would be required if the stand was constructed of aluminum. Steel is also a good option for the prototype, because one of the harder requirements for this mechanism, is that it needs to support a stone slab up to 1500lb. Stone is very brittle and needs to be supported adequately with minimal deflection to reduce the chance of the stone cracking or breaking. Steel provides a high strength that aids in the ability to support the stone with a simple design. This mechanism is being used as a proof of concept, being displayed at a granite tooling show at the end of the 2017-2018 school year. Where the potential marketability of the stand will be assessed and the current design can be modified to solve other problems in the future.

One of the bigger issues that was ran into during the manufacturing process what figuring out how to cut the angled notches in the outer portions of the back legs, as seen in appendix B.20 and B.21. It was decided to use an angle grinder to cut the notches. This was the best way to go about things due to the fact the legs were going to be welded on to the outer vertical post, any imperfections in the cuts could be filled in during the welding process. During the process of welding the legs to the vertical post, the heat from welding deformed the vertical post, making it difficult or impossible for the inner post to slide up and down. The inner post had to be sanded down with a sanding disk as well as a notch milled along the length of the post to make room for the weld bead on the inside of the outer post. One of the changes made to the design of the stand was to add a collar to weld the fine adjustment in to allow for easy disassemble, or the ability to change to a different fine adjustment threaded rod and nut. The majority of the stand was cut and welded together with the rest being pinned together. Another change that had to be made was using a bigger mounted bushing with a taller center height. The original mounted bushing had a center height that was too small and caused the Pivot Plate and the Pivot mount to run into each other and impeded the way the mechanism worked. The new mounted bearings had bigger holes that were spread father apart this was fixed by drilling one of the original holes for the other mounted bushing bigger then drilling an addition hole for the new bushing. The third old hole was then filled in weld and ground flat.

Testing Methods

The slab Tipper has several requirements that could be tested. One of the biggest requirements is that the stand cannot damage the floor when it is under load. Flooring tile is rated to withstand 250lb of force unsupported without breaking.

Test 1:
In appendix A.3 there is an analysis sheet calculating the psi of each foot over a given area. Testing to make sure that the slab tipper matches these numbers is very simple. By placing each
foot of the stand on an individual scale and loading the stand, the downward force that will be exerted on the floor can be assessed.

Test 2:
To ensure that the stand distributes the weight enough to prevent damaging a tile floor, several test floors can be constructed using different types of floor tile, both installed properly and improperly (using the right amount of grout and a separate test floor that doesn’t use the right amount of grout). Then the stands can be placed on the test floors and be loaded up with a full stone slab, observing the floor for cracks of breaks.

Another requirement that NSI Solutions has brought importance is the capability of the stand being used in a 42” aisle.

Test 3:
The mechanism must be able to be used in an aisle 36”-42”. To test its capability in fulfilling this requirement a replica setup can be made using a table and a wall. Placing the table 36”-42” way from the wall, then setting up and loading the stands with a stone, a test can be done to see if the mechanism can transport the stone onto the counter top.

These methods are just some of the ways the mechanism can be tested. Other methods of testing that take place will be added to the report when they take place. NSI Solutions will also have some other testing method ideas that can take place during the time set aside for additional testing.

After the construction process it was clear that the biggest question about the slab tipper is if it is stable enough to hold a slab of stone and transport it onto a counter top. With this in mind the testing plan was changed from a weight distribution test, to a full on mock installation to test the overall functionality of the Slab Tipper as well as its stability and its ability to support a counter top. All of the testing took place in Mukilteo Washington at NSI Solution.

Test 1:
The Stand were set up for a mock install using a shipping crate as a counter, as seen in Appendix G.2. A forklift was used to pick up a slab of stone and set it on the stands. As predicted after construction, the stability of the stands was apparent to be an issue with the functionality of the mechanism. When loaded with a slab of stone of about 400lbs, the stand would pivot of the front foot and would fall over if the load was fully on the stand. Due to safety precautions the first testing process was stopped. After the first test it was decided that the base needs to be increased in size to make it more stable.

With the first test finished it was clear the center of gravity of the mechanism and stone was too far forward for the size of base that the slab tipper had. The changes that were made to the stand was welding a plate to the base of the stand to increase the area, as well as shortening the tipping surface to move the center of gravity back.
Test 2:
After the modifications were made to the stands, the second test could be done. This test was just like the first, to see the setup go to appendix G.3. With the same process a forklift was used to pick up a slab of white quartz that was about 483lbs. This particular stone was selected for the increased strength of the manmade product as well as its consistent density of about 20lbs/ft². When loaded the stands remained stable and the Slab Tipper was able to transport the stone onto the counter top by both 1 and 2 men. This was a big improvement from the first test. With the success with the smaller stone and larger white quartz slab was then loaded onto the slab tipper. The Larger stone was approximately 950lbs. The Slab Tipper remained stable and a single person was able to tip the stone controllably onto the counter top.

All of the testing was overseen by a profession in the Granit industry with over 10 years of experience. After the mock install test the stand were weighed and had a final weight of approximately 100lbs per stand, twice the required limit. The force required to keep the stone horizontal was gathered using a scale. Using the findings a force graph was able to made to show the force required to transport the stone to the counter top both with and without the Slab Tipper. To see the data gathered and the force chart see appendix G.1.

**Budget/Schedule**

NSI Solutions have provided a total budget of $1000, this is the money that they have set aside for this project alone. Additional money may be requested and provided upon approval by NSI Solutions. Current the budget consists of parts and material costs that will be ordered for the manufacturing stage of the project. Also is included in the budget is a personal labor cost $1.25/hour. The total cost of labor depends on the total time it takes in the design, manufacturing, and testing processes. The current budget breakdown chart is located in Appendix D.1, in the figure it shows the cost of materials to be $729.81, the Material cost table is located in Appendix C.1. D.1 also shows the current Total cost of $98.75 and the predicted final cost of $958.87. The predicted cost is under the $1000 budget but methods to reduce the cost of the project should be evaluated, to help prevent going over the budget.

The Slab tipper project was started approximately 30 days late due to complications and availability one certain products necessary to start the construction process. This pushed everything back, the construction process was behind the majority of the winter quarter only catching up and getting on track during the final two weeks of the quarter. One of the reasons why it took so long for the project to get caught up was due to the lack of machine shop and welding shop time. With these setbacks it was still possible to get the project finished on time along with additional improvements to the mechanism. The original projected total time was 183.25 hours with the current time spent on the project of 119.25 hrs. This leave approximately 60hrs for the testing process.

The original projected cost for the materials for the Slab Tipper was approximately $729.81. This was under the $1000 budget, but during the construction process some changes were made to the design of the mechanism as well as material choices changed. After the Tippers were fully construction the material costs came out to be $390.85. This is almost a 50% cost cut.
from the original projected cost, and falls well below the budget limit of $1000. This leave room for additional purchases during the testing process to aid in testing as well as making changes as needed.

During the Testing portion of the project some issues came to light and modifications had to be made, this caused additional purchases of material. The modifications were required to be made to make the mechanism functional. After all of the modifications were made to make the mechanism work the final cost of materials was $460.69, this is well under the $1000 budget and the predicted cost of the project.

Discussion

During the design process of this project, the slab tipper has under gone several revisions throughout the entirety of the fall quarter. The original design ideas consisted of both a free standing 4 legged stand and a stand that required that it be mounted to the cabinet prior to the installation of the stone counter top. The design that required to be mounted to the cabinet was quickly scrapped by NSI Solutions. The four legged design was later changed to a lighter more compact 3 legged design, where most of the design process took place. You can see the three legged design in Appendix B.7. Another idea that was scrapped pretty early in the design process was the use of a linear speed limiter. NSI Solutions decided that there wasn’t a big enough need for the limiter to justify adding it to the design. Appendix B.5 shows the design that incorporated the linear speed limiter. The company that requested the Slab Tipper also requested a simplified design of the stand, which aided in revision 2 of the fine adjustment mount. The original Fine Adjustment Mount is located in Appendix B.10 and revision 2 of the mount is located in Appendix B.11. One of the bigger design changed that greatly reduced the potential cost of this project was getting rid of a third stand to support the middle of the stone. Although the third stand would have been a more rigid method in supporting the center of the stone. A beam design that ran between two stands was chosen to support the center of the stone. This greatly reduced the total weight of the mechanism as well reducing the storage size and cost of the project. The Beam design also went under a couple revisions the first is shown in Appendix B.15, this design didn’t support the center of the stone enough and had a high risk of the stone breaking at the midpoint. After a couple more revisions in the calculations located in Appendices A.10-13, The final design for the middle support is two 2”x1”x8’-3/16” Steel Tube that run the distance between two stands. This design can be seen Appendix B.12.

Conclusion

Success of this project relies on several major requirements, the ability to support a stone slab with a weight of 1500lb, protecting the floor by distributing the weight of the stone enough to prevent damaging the floor, and limiting the amount of workers required for a big installation. Calculations for these requirements can be found in Appendix A. A.13 shows the calculations of the final design for supporting the center of the stone. These calculations show that the stone will have minimal deflection when in the horizontal position limiting the potential of breaking the stone during installation. A.3 and A.18 shows the forces broken down along the components of the stand showing the force that each leg exerts on the floor then being dispersed across the
surface area of the foot to limit the chance of the mechanism damaging the floor. Normally 5-6 workers are required for a big installation, but since the stand is supporting a majority of the stone slab, it is predicted that 2-3 workers may be required for the same install using the Slab Tipper mechanism. This can’t be directly calculated using green sheets but Appendix A as a whole shows a significant amount of proof that this is possible. Assessment of this requirement will be evaluated upon completion of the manufacturing and testing stages.

After the construction process the two stands are complete and ready for the testing process. The stand came together as planned along with changes made after the design process. The stand is on the heavy side and could possibly be over the weight requirement. This is a minor issue due to the fact that the pivot plate and the stand can be separated from each other allowing for it to be moved with ease. The stand allows for installation at all required counter heights and room for variation.

The main requirement of the project was the mechanisms ability to transport a slab of stone to the counter top, this was proven to be met by the testing that was done. Along with this requirement was other such as be able to be operated by 2-3 people, weigh less than 50lbs, protect a tile or wood floor (exert less than 200lbs at any given point onto the floor), able to be used in a 42” isle, counter tops range from 30 to 42 inches, and cost less than $1000. All of the requirements were met except of the weight requirement. The mock install test was able to prove most of these to be met, the tipping surface was shortened to work in a 36” isle, 1 person was able to tilt a 963lb stone. The requirement of protecting the floor was not tested, but the calculations for three feet was done before testing and was predicted to be successful, so after the plate was welded to the feet it was decided that this requirement was met, as the plate was dimensioned to always be over at least one supporting beam in the floor. The weight requirement was not met, and it was about double the 50lb requirement, but this was decided to be the lease important of the requirements, as it is only a proof of concept for the company. If this was to be a product adopted by NSI Solutions the final product would be made of aluminum. Even with this in mind the tipping portion of the mechanism is easily separated from the stand portion, the base with plate weighs approximately 55lbs and the tipping portion about 40lbs, this makes the weight considerably easier to handle. After the whole testing portion was completed with modifications the project was deemed a success. All primary requirements were met, and the owners of NSI Solutions are satisfied with the Mechanism, it shows plenty of room for improving, and a promising future as having potential to become a marketable product.
Appendices

Appendix A.1 – Force Distribution on Stand
Appendix A.2 – RADD Weight Distribution 4 Legs

RADD

Requirement: When supporting a full finished piece of stone the stand must not damage the floors. Tiles have a breaking strength of 250 lb, no tile should ever experience 250 lb x force.

ASTM C648-04
Outrigger Pad

Analysis:

Avg max stone weight = 1500 lb
SF = 2.0

Avg weight x S.F. => 1500 lb x 2.0 = 3000 lb

\[ \Sigma F_x = 0 \Rightarrow \Sigma F_y = 0 \]
\[ 3000 \text{ lb} - F_{s_x} (x) = 0 \Rightarrow F_{s_x} (x) = 3000 \text{ lb} \]

\[ F_{s_x} = 1500 \text{ lb} \]

\[ F_{s_y} = 0 \Rightarrow F_{s_y} = 1500 \text{ lb} \]

\[ F_{s_x} = F_{s_y} \]
\[
F_{x1} = -F_{x2} \implies 0
\]
\[
F_{y1} = F_{y2}
\]
\[
\sum F_y = 0: \quad 1500 \text{lb} - 2F_{y1} = 0 \implies 2F_{y1} = 1500 \text{lb}
\]
\[
F_{y1} = 750 \text{ lb}
\]
\[
F_{y2} = 750 \text{ lb}
\]
\[ F_1 = F_2 \]

\[ \Sigma F_y = 0 \quad 750 \text{lb} - 2F_1 = 0 \Rightarrow 2F_1 = 750 \text{lb} \]

\[ F_1 = \boxed{375 \text{ lb}} \]

\[ F_2 = \boxed{375 \text{ lb}} \]

375 lb is the force that each leg will be pressing down on the floor.
Appendix A.3 – RADD #2 Weight Distribution 3 Legs

Given:
- Requirement: Stand shall distribute weight enough to prevent the floor from breaking.

ASTM C648-04 250 lb
Max Stone weight = 1500 lb
SF = 2.0

Solution:

\[ F_{S_1} = F_{S_2} \]

\[ E_{fy} = 0.1 \times (2.0) \times (1500 \text{lb}) - (2) \times F_{S_1} \]

\[ F_{S_1} = \frac{3000 \text{lb}}{2} \]

\[ F_{S_1} = 1500 \text{lb} = F_{S_2} \]
Given:

10.06
10.99
23.57

Find: Forces in each leg

Solution:

\[ F_{y} = 0 \]
\[ 1500 \text{ lb} - 3 F_{L} \cos 25^\circ = 0 \]
\[ F_{L} \cos 25^\circ = \frac{1500 \text{ lb}}{3} \]
\[ F_{L} = 551.69 \text{ lb} \]

Assuming all legs are at exactly 25° from vertical

\[ F_{L1} = F_{L2} = F_{L3} = 551.69 \text{ lb} \]
Given: Force exerted on floor by one individual leg

Find: Force exerted on floor by one individual leg

Solution: \[ \sum F_y = 0: \quad 651.0 \cos 25^\circ - F_i = 0 \]

* Don't need to solve for \( F_s \) (force friction)

\[ F_i = 500 \text{ lb} \]
Given: Vertical Point Force = 500lb on Floor
ASTM C 1748-04
Floor breaks @ 250lb

Find: Minimum area of foot to not Break Tile on floor

Solution:
Since the forces at hand are point forces as long as the weight is distributed over any area the tile should not Break.
Appendix A.4 – Buckling and Tube Size

Using Buckling forces to Calculate Size of A36 Tubing needed

(Number for forces are from RADD #1)

Given:
- \( F = 1500 \text{ lb} \) (with a SF of 2.0)
- Material: A36 Hot Rolled Steel tubing
- \( E = 30 \text{ Msi} \)
- 2 fixed ends

Find: Cross sectional Area of Tube needed

Solution:

\[ F = \frac{\pi^2 EI}{(KL)^2} \implies \frac{F(KL)^2}{\pi^2} = I \]

\[ I = \frac{1500 \text{ lb} [0.5 \text{ in}]}{30,000 \text{ ksi} \times \left( \frac{24 \text{ in}}{1 \text{ in}} \right)^2} = 0.0007295 \]
Buckling with one Pivot and one fixed end.

\[ F = \frac{\pi^2 EI}{(KL)^2} \]

\[ I = \frac{F(KL)^2}{\pi^2 E} \]

\[ I = 0.00292 \]
Finding Tube Size

Area Moment of Inertia Section Properties = I
\[ I = \frac{a^4 - b^4}{12} \]

\( \text{C 2'' x 2'' - } \frac{1}{8} \text{ thick Tubing} \)
\[ a = 2'' \]
\[ b = 1.75'' \]
\[ I = \frac{2^4 - 1.75^4}{12} = 0.5518 \]

\( \text{C 1.75'' x 1.75'' - } \frac{1}{8} \text{ thick Tubing} \)
\[ a = 1.75'' \]
\[ b = 1.5'' \]
\[ I = \frac{1.75^4 - 1.5^4}{12} = 0.8597 \]
Appendix A.5 – Pin Shear

Finding Pin Size

Given:
- Force of leg onto pin is 375 lb
  (using numbers from RADD+1)
  * SF. = 2.0*

Find: Pin size

Solution:
18-8 Stainless Steel Clevis Pin
5/16” Diameter, 2-1/8” usable length

Shear Stress $\sigma_{avg} = \frac{F}{2\pi r^2}$

$\Rightarrow 375 \text{ lb} \frac{2\pi (0.15625)^2}{2} = 2441.62 \text{ psi}$

Bearing Stress
- Foot $B_f = \frac{F}{2t_d}$
  $\Rightarrow 375 \text{ lb} \frac{2(\frac{1}{8})(\frac{1}{16})}{2} = 4800 \text{ psi}$
- Leg $B_l = \frac{F}{td}$
  $\Rightarrow 375 \text{ lb} \left[\frac{1}{8}\right]\left[\frac{5}{16}\right] = 9400 \text{ psi}$
Appendix A.6 – Fine Adjustment Buckling Rev1

Given: Steel threaded Rod

- F = 750 lb
- D = 1 1/2" - 6 Grade B
- SF = 2.0
- Fixed one end
- L = 8.00"
- k = 2.10
- Sy = 130 ksi
- E = 80 Msi

Find: Critical load & Allowable load

Solution:

\[ A = \frac{\pi D^2}{4} \Rightarrow \frac{\pi (1.25)^2}{4} = 1.23 \text{ in}^2 \]

\[ I = \frac{\pi D^4}{64} \Rightarrow \frac{\pi (1.25)^4}{64} = 0.1198 \text{ in}^4 \]

\[ r = \sqrt{\frac{I}{A}} \Rightarrow \sqrt{\frac{0.1198 \text{ in}^4}{1.23 \text{ in}^2}} = 0.3125 \text{ in} \]

Slenderness Ratio \( \frac{kL}{r} \Rightarrow \frac{(2.10)(8.00 \text{ in})}{0.3125 \text{ in}} = 53.76 \]

\[ C_c = \frac{2\pi^2 E}{Sy} \Rightarrow \sqrt{\frac{2\pi^2 (30 \times 10^6)}{130 \times 10^6}} = 61.49 \]

Johnson Formula

\[ P_{cr} = AS_y \left[ \frac{Sy(1/2)}{4\pi^2 E} \right] = (1.23 \text{ in}^2)(130 \times 10^6) \left[ \frac{4\pi^2 (30 \times 10^6)}{130 \times 10^6} \left( \frac{2.10}{0.8125 \text{ in}} \right)^3 \right] \]

\[ P_{cr} = 109174.15 \text{ lb} \]

\[ P_a = P_{cr}/SF = \frac{109174.15 \text{ lb}}{2.0} = 54587.07 \text{ lb} \]
Appendix A.7 – Vertical Adjustment Buckling

Given:
- $F = 750\text{ lb}$
- $SF = 2.0$
- Fixed one end
- $L = 10.25\text{ in}$
- AISI steel tube
- $k = 2.10$
- $E = 30\text{ KSI}$
- $S_y = 30\text{ KSI}$

Find:
- Critical load $P_c$
- Allowable load

Solution:

1. Calculate $A$:
   \[ A = BH - bh = (1.75 \times 1.75) - (1.5 \times 1.5) = 0.8125\text{ in}^2 \]

2. Calculate $I$:
   \[ I = \frac{BH^3 - bh^3}{12} = \frac{(1.75)(1.75)^3 - (1.5)(1.5)^3}{12} = 0.3897\text{ in}^4 \]

3. Calculate $R$:
   \[ R = \sqrt{\frac{0.8125\text{ in}^2}{0.3897\text{ in}^4}} = 0.8125\text{ in} \]

4. Calculate $k_L$:
   \[ k_L = \frac{1}{0.8125\text{ in}} \]

5. Calculate Slenderness Ratio:
   \[ \text{Slenderness Ratio} = \frac{(2.10)(10.25)}{0.8125\text{ in}} = 32.351 \]

6. Calculate Column Constant $C_c$:
   \[ C_c = \sqrt{\frac{2\pi^2 E}{S_y}} = \sqrt{\frac{2\pi^2 \left(30 \times 10^5\right)}{36 \times 10^6}} = 128.25 \]

7. Apply Johnson Formula:
   \[ P_c = \frac{A S_y (k_L/6)^2}{4\pi^2 E} \Rightarrow 1 - \frac{(23.16\times 10^3)(0.126)(10.25)}{4\pi^2 \left(30 \times 10^5\right)} \]
   \[ P_c = 2831.96\text{ lb} \]

8. Calculate Allowable Load:
   \[ P_a = \frac{P_c}{SF} = \frac{2831.96\text{ lb}}{2.0} = 1415.98\text{ lb} \]
Appendix A.8 – Fine Adjustment Buckling Rev 2

Given:
1"-8 Grade 8 Threaded Rod
- F = 750 lb
- SF = 2.0
- Fixed one end
- L = 8.00"
- k = 2.10
- Sy = 130 Ksi
- E" = 30 Msi

Find:
Critical load & Allowable load

Solution:
- \( A = \frac{\pi D^2}{4} \) = 0.519 in^2
- \( I = \frac{\pi D^4}{64} \) = 0.0214
- \( R = \sqrt{\frac{I}{A}} = \sqrt{\frac{0.0214}{0.519}} = 0.203 \)

Slenderness ratio:
- \( \frac{kL}{R} = \frac{2.10 \times 8.00}{0.203} = 82.66 \)

\( C_o = \sqrt{\frac{2\pi^2 E}{Sy}} = \sqrt{\frac{2\pi^2 (30 \times 10^6)}{(130 \times 10^2)}} = 67.49 \)

Euler formula:
- \( P_{cr} = \frac{\pi^2 E A}{(kL/R)^2} = \frac{\pi^2 (30 \times 10^6) (0.519 \text{ in}^2)}{2.10 \times 8.00^2} = 22,497.51 \text{ lb} \)
- \( P_a = \frac{P_{cr}}{SF} = \frac{22,497.15 \text{ lb}}{2.0} = 11,248.76 \text{ lb} \)
Appendix A.9 – Fine Adjustment Buckling Rev 3

Given:
- Threaded Steel Rod
  - F = 1760 lb
  - SF = 2.0
  - Fixed one end
  - L = 8.00 in
  - k = 2.10
  - S_y = 130 ksi
  - E = 30 Msi

Find:
- Critical load and Allowable load
  - 750 lb

Solution:
- \[ A = \frac{\pi D^2}{4} \Rightarrow \frac{\pi (0.600)^2}{4} = 0.283 \text{ in}^2 \]
- \[ I = \frac{\pi D^4}{64} \Rightarrow \frac{\pi (0.600)^4}{64} = 0.00681 \text{ in}^4 \]
- \[ r = \sqrt{\frac{I}{A}} \Rightarrow \sqrt{\frac{0.00681 \text{ in}^4}{0.283 \text{ in}^2}} = 0.15 \text{ in} \]
- \[ \text{Slenderness Ratio} \quad \frac{kL}{r} \Rightarrow \frac{(2.10)(8.00)}{0.15} = 112 \]
- \[ C_e = \sqrt{\frac{2\pi^2 E}{S_y}} \Rightarrow \sqrt{\frac{2\pi^2 (30 \times 10^6)}{(130 \times 10^3)(0.15)}} = 6.749 \]
- \[ P_{cr} = \frac{\pi E A}{(KL/r)^2} \Rightarrow \frac{\pi^2 (30 \times 10^6)(0.283 \text{ in}^2)}{(2.10)(8.00)^2} = 1513.8 \text{ lb} \]
- \[ P_a = \frac{P_{cr}}{SF} \Rightarrow \frac{1513.8 \text{ lb}}{2.00} = 333.0 \text{ lb} \]
Appendix A.10 – Middle Support Beam Bending Rev 1

Given:

- ASTM A36 Grade A
- E = 30,000 ksi
- 500 lb/ft applied to beam
- Beam length = 9 ft
- Middle support beaming

Find: Max deflection

Solution:

\[ I = \frac{\pi}{32}(D^4 - d^4) \]

\[ I = 0.125(1.25^4 - 1.0^4) = 0.025 \]

\[ y_{max} = \frac{PL^3}{48EI} \]

\[ y_{max} = \frac{(250 \text{ lb})(9 \text{ ft})^3}{48(30 \times 10^6)(0.025)} = 5.46 \text{ in} \]
Appendix A.11 – Beam Bending Middle Support Rev 2

Given:

\[ E = 10 \text{ Msi} \]

Aluminium

Find: Max deflection

Solution:

\[ y = \frac{5WL^3}{384EI} \]

\[ I = \frac{6H^3 - bh^3}{12} \Rightarrow \frac{1(2^3) - (0.75)(1.75^3)}{12} = 0.332 \]

\[ y_{max} = \frac{-5(2501b)(96)}{(384)(10\times10^5)(0.332)} = -0.868 \text{ in} \]

Too much deflection
Appendix A.12 – Beam Bending Middle Support Rev 3

Given:
- Steel
- E = 30 Msi

Find: Max deflection

Solution:

\[ I = \frac{bh^3}{12} \]

\[ \Rightarrow \frac{(1)(2)^3 - (0.75)(1.75)^3}{12} = 0.332 \]

\[ y_{max} = \frac{5WL^3}{384EI} \]

\[ \Rightarrow \frac{(-5)(250)(96)^3}{(384)(30 \times 10^6)(0.332)} = 0.289 \text{ in} \]

\[ \approx 0.28 \text{ in} \]
Appendix A.13 – Beam Bending Middle Support Rev 4

Given:

\[ \begin{align*}
\text{250lb} & \quad 9\text{in} \\
\end{align*} \]

Find: max deflection

Solution:

\[ \begin{align*}
I &= \frac{bh^3}{12} \\
\frac{250 \times 2\text{in}}{12} &= \frac{(1\times 2)^3 - (1.625)(1.625)^3}{12} = 0.4482 \\
Y_{\text{max}} &= \frac{-SWL^3}{384EI} \\
&= \frac{(-5)(250)(9\text{in})^3}{384 (30 \times 10^6)(0.4482)} = -0.2166\text{in}
\end{align*} \]

.2166 in
Appendix A.14 – Pivot Bracket Beam Bending

Given:
- L = 375 lb
- E = 30,000 ksi
- A36 Steel Tube

Find: Total Deflection

Solution:

\[ V_{\text{max}} = \frac{PL^3}{4BEI} \]

\[ I = \frac{bh^3}{12} \]

\[ I = \frac{(2.00)(1.00)^3 - (1.75\text{in})(0.75\text{in})^3}{12} = 108 \]

\[ V_{\text{max}} = \frac{(750\text{lb})(11.63\text{in})}{48(30\times10^6)(108)} = 0.0078 \text{ in} \]
Appendix A.15 – Foot Pin Shear Force

Given:

- Foot Pin Shear
- 1/4" A36 Clevis pin

Double shear

Solution:

\[ T = \frac{F}{A} \]

\[ T = \frac{551.69 \times \pi (0.5)^2}{2} \]

275.85 lb

275.85 lb

275.85 lb

Find: Shear in pins in foot

\[ F = 551.69 \text{ lb} \]

\[ \text{1404.87 lb} \]
Appendix A.16 – RADD #3 Pivot Pin Shear

Given:
- Requirement: Must be able to support a slab weight of 1500 lb
  - 375 lb Force
  - Single Shear
  - 5 in

Find:
- Shear Force in pin to figure out pin material

Solution:
\[
T = \frac{F}{A} \cdot \frac{375 \text{ lb}}{100 \text{ in}} = 1909.8 \text{ psi}
\]

* Basic Mild A36 1/2" - 2" clevis pin will work
Appendix A.17 – Vertical Adjustment Pin Shear

Find: Shear in pin to ensure \( \frac{1}{2} \) A36 clevis pin will work

Solution:

\[
T = \left( \frac{F}{A} \right) / 2 = \left( \frac{750}{\pi (\frac{5}{8})^2} \right) / 2 = 1909.85 \text{psi}
\]

* \( \frac{1}{2} \) A36 clevis pin will work
Appendix A.18 – Pressure Exerted on Floor by the Foot

Given:
- 500 lb
- 5 in
- 2 in

Find:
Psi the Foot exerts on the Floor

Solution:
\[ A = BH = 2 \times 5 = 10 \text{ in}^2 \]

\[ \frac{500 \text{ lb}}{10 \text{ in}^2} = 50 \text{ psi} \]

Since the tile can hold up to 280 lb at any given point, 50 psi exerted on the floor should not break the tile.
Appendix A.19 – Shear of Adjuster Pin

Given:
- Adjustable Pin Shear
- 375 lb
- A36 Steel
- E = 30 Msi

Find: Shear force in pin to find material

Solution:

\[ T = \frac{F}{A} \implies \frac{375 \text{ lb}}{0.194 \text{ in}^2} = 1909.9 \text{ psi} \]

\[ A = \frac{\pi D^2}{4} \implies \frac{\pi (0.5)^2}{4} = 0.194 \text{ in}^2 \]

*Basic A36 Steel will work*
Appendix A.20 – Bolt Shear

Given:

- Bolt: 7/16" Grade 8, 760 ksi
- 1500 lb
- 187.5 lb
- To find the shear in the bolt,

Solution:

\[ T = \frac{F}{A} = \frac{187.5 \text{ lb}}{\frac{\pi (7/8)^2}{4}} = 6754.54 \text{ lb/in}^2 \]

91 ksi for Grade 8 bolts.

High-Strength Grade 8 Steel Hex Head Screw
Zinc-Aluminum Coated 7/16"-20 2-1/2" long

9/28/12 A123
Appendix A.21 – Beam Bending Pivot Plate
\[ y = \frac{PL^2}{8EI} \]
\[ y = \frac{93.7^{1/6}(30^2)}{3(30 \times 10^4)(.2884)} \]
\[ y = 0.0975 \text{ in} \]

\[ y = \frac{4.69^{1/6}}{8EI} \]
\[ y = \frac{-WL^3}{3EI} \]
\[ y = \frac{-4.69(30)(30^3)}{8(30 \times 10^4)(.2884)} \]
\[ y = 0.05406 \text{ in} \]

\[ y = 0.0549 + 0.0975 = 0.1524'' \]
Appendix B.1 – stand foot Revision 1
Appendix B.2 – Ladder Style Foot
Appendix B.4 - Support Plate
Appendix B.5 – Linear Speed Limiter
Linear Speed Limiter Mounting Method #1
Appendix B.7 – Stand Assembly
Appendix B.8 – Exploded Stand Assembly
Appendix B.9 – Bent Leg Plate
Appendix B.11 – Fine Adjustment Assembly Rev 2
Appendix B.12 – Full Assembly Rev 2

There will be another stand at this end.
Appendix B.13 – McMaster Carr ½” – 2” Steel Clevis Pin
Appendix B.16 – Assembly Drawing

Stand Assembly

SIZE: DNG. NO. REV

SCALE: 1:24 WEIGHT: SHEET 1 OF 1

Part Number | Part Name                      | Specifications
-------------|--------------------------------|-------------------
1            | Outer Vertical Adjustment Tube | 2”x2” Steel Tub
2            | Bent Leg Plate                | A36 Steel Plate 125”
3            | Fixed Rear Leg Outer          | 2”x2” Steel Tube
4            | Pivoting Front Leg            | 2”x2” Steel Tube
5            | Front Leg Pivot Brace         | A36 Steel Plate 125”
6            | Rear Inner Leg                | 1.75”x1.75” Steel Tube
7            | Foot                          | A36 Steel Plate 125”
8            | Course Adjustment Tube        | 1.75”x1.75” Steel Tube
9            | Fine Adjustment Rod           | 1”-8 1” Threaded Steel Rod
10           | Fine Adjustment Nut           | 1”-8 Steel Coupling Nut
11           | Face Plate                    | 1”x1” Steel Tube
12           | Pivot Bearing                 | 1/2” Oil Embedded bearing
13           | Adjuster Pin                  | 1” A36 Round Stock
14           | Clevis Pin                    | 1/2”x2” Steel Clevis Pin
15           | Clevis Pin                    | 1/2”x2” 1/2” Steel Clevis Pin

All Pins for Legs and vertical course adjustment will be 1/2”x2 1/2” Steel Clevis Pins

UNLESS OTHERWISE SPECIFIED:
DIMENSIONS ARE IN INCHES
FRACTIONAL ANGULAR MACH: 1” BEND ± TWO PLACE DECIMAL ± THREE PLACE DECIMAL
INTERPRET GEOMETRIC TOLERANCING PER
MATERIAL

NAME
DATE
DRAWN
CHECKED
ENG APPL
MFG APPL
Q.A.
COMMENTS

APPLICATION
DO NOT SCALE DRAWING

PROPRIETARY AND CONFIDENTIAL
THE INFORMATION CONTAINED IN THIS DRAWING IS THE SOLE PROPERTY OF <INSERT COMPANY NAME HERE>. ANY REPRODUCTION IN PART OR AS A WHOLE WITHOUT THE WRITTEN PERMISSION OF <INSERT COMPANY NAME HERE> IS PROHIBITED.
Appendix B.17 – Foot Design Rev 3
Appendix B.18 – Tree

- Slab Tipper
  - Stand
    - Legs
      - Outer and inner Leg
        - Feet
    - Course Adjustment
      - Outer and Inner Course Adjustment Tube
  - Fine Adjustment
    - Face Plate Mount
    - Threaded Rod
  - Face plate
    - Adjustment pin
    - Pivot Bearings
Appendix B.22 – Fine Adjustment Collar

Fine Adjustment Collar

DIMENSIONS ARE IN INCHES

UNLESS OTHERWISE SPECIFIED:

NOTE: PROPERTIES AND DIMENSIONS SHOWN DO NOT CONSTITUTE A CONTRACT BUT ARE SUBJECT TO CHANGE WITHOUT NOTICE. CHECK SPECIFICATION SHEET ORalled WITHOUT NOTICE AND MAY BE SUBJECT TO CHANGE WITHOUT NOTICE. CHECK SPECIFICATION SHEET OR

REV.

DCW.

D.0007

SCALE: 1:1

WEIGHT:

SHEET 1 OF 1

SAVE DWG.

NOTE:

MATERIAL

Q.L.

COMMENTS:

DRAWN

CHECKED

ENG. APPRO.

ENGINEERED

DATE

NAME

D.M. EFFECTIV.

NEXT ASY.

USED-ON

APPLICATION
Appendix B.23 – Pivot Mount Assembly
Appendix B.26 – Photos of Construction Process
Appendix B.26 – Modifications
Appendix B.27 – Video

Construction:

https://www.youtube.com/watch?v=bJZ7zSVKmVM&feature=youtu.be

Testing:

https://www.youtube.com/watch?v=dxC4jQtkDOY
## Appendix C.1 – Cost Sheet

### Cost Of Material and Parts

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<tr>
<th>Material/Part</th>
<th>Company</th>
<th>Price each ($)</th>
<th>Quantity</th>
<th>Total for Part ($)</th>
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<tbody>
<tr>
<td>Fixed Bearing</td>
<td>McMaster Carr</td>
<td>11.11</td>
<td>4</td>
<td>44.44</td>
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<tr>
<td>1&quot; - 8 Threaded Rod 1'</td>
<td>McMaster Carr</td>
<td>11.63</td>
<td>2</td>
<td>23.26</td>
</tr>
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<td>McMaster Carr</td>
<td>8.99</td>
<td>4</td>
<td>35.96</td>
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<td>148.26</td>
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<td>2</td>
<td>32.6</td>
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<td>2&quot;x2&quot;x120&quot;Steel Tube</td>
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| Total Combined Price                       |                  |                |          | 729.81            |

### Actual Cost Sheet

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<th>Item</th>
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<th>Part Number</th>
<th>Quantity</th>
<th>Cost ($)</th>
<th>Date Ordered</th>
<th>Date Received</th>
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</thead>
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<td>1/2&quot; Oil-embeded Mounted Sleeve Bearing</td>
<td>McMaster Carr</td>
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<td>5912X4</td>
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<td>44.44</td>
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# Appendix D.1 – Budget

## Budget

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## Labor

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**Predicted Final Total Cost** 958.87

**Current Total cost** 98.75
Appendix E – Schedule Gantt chart

Below is a Gantt Chart breaking down the Slab Tipper project over the course of the year into Fall, Winter, and Spring Quarters. Fall Quarter consists of the design process of the project as well and the green sheet calculations and writing of the proposal. Winter Quarter consists of the Manufacturing stage of the project and Spring Quarter is the testing portion of the project. You can view each quarter in depth in the chart below breaking down each phase into fine detail. Milestones are identified by ◊.

To View Gantt Chart in Greater Detail zoom in using slider located in the bottom right corner of word.
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</table>
Appendix F - Acknowledgment

Project was provided and requested by NSI Solutions

Funded by NSI Solutions

Mentors for Advice
  Don Durand – NSI Solutions
  Dr. Johnson – CWU
  Professor Pringle – CWU

Facilities Provided by
  NSI Solutions
  CWU Shop

Winter Quarter:

Thank you to Staff for monitoring the CWU Shop and Welding Shop open for the manufacturing process as well as providing help when needed. There was several parts during the manufacturing process that required the aid of another person and the CWU Shop Techs and Staff were there to provide the assistance. Thank you to Don Durand from NSI Solutions for providing funding for the project, advise and assisting in the ordering process.
Appendix G.1 – Testing Data

<table>
<thead>
<tr>
<th>Slabs</th>
<th>Length (ft)</th>
<th>Width (ft)</th>
<th>weight (lbs)</th>
<th>Type</th>
</tr>
</thead>
<tbody>
<tr>
<td>Stone 1</td>
<td>4.17</td>
<td>7</td>
<td>583</td>
<td>White Quartz</td>
</tr>
<tr>
<td>Stone 2</td>
<td>5.5</td>
<td>8.75</td>
<td>963</td>
<td>White Quartz</td>
</tr>
</tbody>
</table>

<table>
<thead>
<tr>
<th>Weight Test</th>
<th>Base weight (lbs)</th>
<th>Tipping surface (lbs)</th>
<th>total weight (lbs)</th>
<th>requirement weight (lbs)</th>
<th>percent over</th>
</tr>
</thead>
<tbody>
<tr>
<td>Slab Tipper V2</td>
<td>53</td>
<td>40</td>
<td>93</td>
<td>50</td>
<td>186%</td>
</tr>
<tr>
<td></td>
<td>106%</td>
<td>80%</td>
<td></td>
<td></td>
<td></td>
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</table>

<table>
<thead>
<tr>
<th>force to hold stone horizontal</th>
<th>Projected for 1500lbs</th>
<th>Actual for 963lbs</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td>90lbs</td>
<td>120lbs</td>
</tr>
</tbody>
</table>

Improvements to be made:

1. make the stand shorter
2. lock in the upright and horizontal position
3. new pins
4. stop from spinning
5. rate control?

Force Required to Transport Slab Onto Counter Top (White Quartz)
Appendix G.2 – Test 1 Setup
Appendix G.3 – Test 2 Setup
## Appendix H – Data Evaluation

### Stone Sizes

<table>
<thead>
<tr>
<th>Width (in)</th>
<th>Length (in)</th>
<th>weight (lbs)</th>
<th>Force required by workers (lbs)</th>
</tr>
</thead>
<tbody>
<tr>
<td>50</td>
<td>84</td>
<td>583</td>
<td>65</td>
</tr>
<tr>
<td>60</td>
<td>100</td>
<td>833</td>
<td>-78</td>
</tr>
<tr>
<td>70</td>
<td>120</td>
<td>1167</td>
<td>-343</td>
</tr>
<tr>
<td>78</td>
<td>132</td>
<td>1430</td>
<td>-643</td>
</tr>
</tbody>
</table>

<table>
<thead>
<tr>
<th>lbs/in (width)</th>
</tr>
</thead>
<tbody>
<tr>
<td>11.67</td>
</tr>
<tr>
<td>13.89</td>
</tr>
<tr>
<td>16.67</td>
</tr>
<tr>
<td>18.33</td>
</tr>
</tbody>
</table>

<table>
<thead>
<tr>
<th>stone above pivot (in)</th>
<th>Weight above pivot (lbs)</th>
<th>weight below pivot (lbs)</th>
</tr>
</thead>
<tbody>
<tr>
<td>23.5</td>
<td>274</td>
<td>309</td>
</tr>
<tr>
<td>33.5</td>
<td>465</td>
<td>368</td>
</tr>
<tr>
<td>43.5</td>
<td>725</td>
<td>442</td>
</tr>
<tr>
<td>51.5</td>
<td>944</td>
<td>486</td>
</tr>
</tbody>
</table>

<table>
<thead>
<tr>
<th>stone above pivot (in)</th>
<th>Weight above pivot (lbs)</th>
<th>weight below pivot (lbs)</th>
</tr>
</thead>
<tbody>
<tr>
<td>23.5</td>
<td>274</td>
<td>309</td>
</tr>
<tr>
<td>33.5</td>
<td>465</td>
<td>368</td>
</tr>
<tr>
<td>43.5</td>
<td>725</td>
<td>442</td>
</tr>
<tr>
<td>51.5</td>
<td>944</td>
<td>486</td>
</tr>
</tbody>
</table>
\[ \sum F_y = 0 \]

\[ A_y = 963.66 + B_y \]

\[ \sum M_B = 0 \]

\[ \frac{(27'')(14.53 \text{ ft})}{1} (27'') + \frac{(39'' + 27'')}{2}(14.57 \text{ ft}) \times (57.7) = A_y \]

\[ \frac{(27'')}{(27'')} \]

\[ A_y = 963.66 + B_y \]

\[ (1176.9 \text{ lb} - 963.66) = 213.2 \text{ lb} \]
Stone: 66' x 105

96.8 lb

963 lb (main) = 14.57 lb

14.37 lb x 29 = 516.8 lb
14.37 lb x 26 = 370.6 lb

M_A = 0: 576.8 lb (\frac{39.5}{2}) - 40 (10.25) - 386.64 (\frac{26.5}{2})

P = 11,382.02 - 2(46) - 5/22.78

P = 205 lb
Appendix I - Slab Tipper Testing

Introduction:
The requirements that will be tested is the mechanisms ability to support a slab of stone and transport it to a counter top. The stone could be a wide variety of weights but it has a maximum weight of 1500lbs. The other things that are going to be tested is the weight of each stand, the force to keep the stone horizontal, and the functionality of the mechanism.

Requirements:
- Mechanism supports and transports a stone up to 1500lbs to a counter top
- Weighs under 50lbs
- can be used by 2-3 people

This testing report includes two testing procedures. For the first test the Slab Tipper was predicted to support the stone and transport it to the counter top, but be slightly unstable due to the three legged construction. It was also predicted to be slightly over the 50lb limit. For testing procedure two the Slab Tipper was predicted to be more stable while holding and transporting the stone, be easier to use, but be significantly over the weight limit.

Methods/Approach:
The testing will be held in Mukilteo at NSI Solutions shop. The owners and workers there will be at hand to help the testing the process. The stone will be provided by the partner company Natural Stone Interiors. The tools needed to conduct the tests are clamps, Fork Lift, Table, and a Scale. The ability to support the stone as well as functionality will be judge by works that have worked in the field for a significant amount of time. The weight will be gathered with an electronic package scale. Stone slabs will be transported and lowered on to the Slab tipper with fork lifts to begin the testing phase. They will then be fully supported by the Slab Tipper. Then a mock installation process will be done by tilting the stand till the stone touches the counter top, this test the stands ability to hold and transport the stone as well as the functionality and practicality of this mechanism. After the stone is horizontal and resting on the counter top a scale will be placed on the end not resting on the counter. The force will be applied in the downward direction till the stone is fully suspended, the scale will then read the force required by the worker to keep the stone at this horizontal position. All of the testing will be recorded via cell phone video/camera.

Testing Procedure:
Test 1
During the first test day the stands were set up next to a shipping crate at counter height. A stone slab was then picked to do the test. When the slab was lowered onto the stands using the fork lift, it was clear that the testing had to be stopped, the stands weren’t stable enough to hold the stone. When under load the stands would pivot over the front foot and was very unstable. If the testing were to continue it would be putting the testers at risk of injury. Then a meeting was held to improvements that need to be made to the stand to increase the stands stability and functionality.

Test 2
Like test 1 the stands were set up next to a table for a mock installation. The difference between tests 1 and 2 was there was modifications made to the stand the biggest one being, a metal plate
was welded to the feet to make one big base, making the base wider and longer, in turn making the stands more stable. The stands were first loaded up with a relatively small slab of 560lbs. With the improvements made to the stands it was able to support the stone without issues. Then two people tilted the stone onto the counter top to get the feel of how the weight transfers during the tilting motion. Then a single person tilted the stone into the horizontal position. Then the stone slab was slid onto the counter top to continue the mock install. It was clear that the stands could handle the weight so a larger stone of 960lbs was selected to another mock install. The same procedure took place, first two people tilted the stone, then one person. After the mock install was done a scale was placed on the end of the stone away from the table. A force was applied in the downward direction on the scale till the stone was fully suspended horizontally. The scale reading was then recorded, this is to figure out the force required by the user to counteract the tipping motion.

**Deliverables:**

After the two tests some conclusions could be drawn about the Slab Tipper. Although the maximum weight of stone was not used during the test, it is clear that the stands done have a problem with holding a stone of approximately 1000lbs. Further testing can be done to test the maximum weight of stone. One of the requirements is that the slab tipper could be used by 2-3 people. This requirement was put into place because the Tipper’s job is to reduce the amount of workers needed to install, as well as making it safer and easier. The with the slab tipper 2-3 people requirement is definitely possible. It is clear that the Tipper makes it easier to get the stone onto the counter top, now the main limiter is getting the stone onto the stand. Testing made it clear that two people could easily tip the slab onto the counter for installation, but most likely two people couldn’t lift the stone onto the tipper without help. This means you only need the number of workers required to lift the stone for the install, this means the number of workers could vary depending on the strength of the given workers. With the 980lb stone the force required to hold the stone horizontal was approximately 120lbs. This means if one worker could get the stone onto the tipper they could do the whole install themselves. The complete tipper weighs 93lbs per stand. This is almost twice the requirement. This is due to the modifications that were required to make the stands more stable. Although the stands are overweight they split into two parts making them easier to transport. These stands are also being used as a proof of concept with the final version being made of aluminum. Over all the Slab Tipper is a success, it fulfills the main requirements and has the capability to be improved to make it a better project. Some of the improvements that could be made to the tippers are installing a locking mechanism to lock the stand in the upright and horizontal position, making the stands shorter and lighter, changing the pins that hold the weight of the stone to have a bigger lip, and preventing the tipping surfaces from spinning when horizontal.
Appendix J - Resume

Thomas Durand

9011 62nd Drive NE
Marysville, WA 98270

Mobile: 425-239-7549
e-mail: Durand.thomas29@gmail.com

Career Objective
A position as a Mechanical Engineer, that will allow me to use my problem solving and design skills I’ve gathered to assist in the design and fabrication of mechanical components.

Education Background
Bachelor of Science, Central Washington University, expected June 2018
   Major: Mechanical Engineering Technology
   GPA: 2.750

Work Experience
Fabrication, December 2016-Present
NSI Solutions, Lynnwood, Washington

Tool Assembly, August 2016-Present
NSI Solutions, Lynnwood, Washington
   • Designing tables, tool displays, etc.
   • Selecting appropriate materials for various projects
   • Welding and building tables, displays and equipment for the shop

Lift Operator, 2017 Winter Season
Summit at Snoqualmie
   • Performed routine lift inspections
   • Excelling in customer service for all patrons of the Summit
   • Operated a high-speed fixed-grip lift

Cashier, March 2015 – December 2017
Central Washington University Dining, Ellensburg Washington
   • Cashier
   • Help customers
   • Stock shelves
   • Moving stock with pallet jack

Skills
• Computer Experience: Excel, Word, PowerPoint, SolidWorks, Auto CAD, CNC manual programming
• Machining: Manual mill and lathe, CNC mill and lathe, Welding, numerous metal and woodworking machines and tools
• CAD: Currently in the process of acquiring my CSWP (Certified SolidWorks Professional)

Achievements
2013-2016 Central Washington University Baseball Team
2013 Native American Student of the Year
2013 Marysville Pilchuck High School Honor Society

References Available Upon Request